**Amendments to the Specification**

Please amend the following paragraphs to read as follows:

[0006] The present invention provides a tail on the non-coupling end of both a male and female member of a hydraulic coupling device. The tail is machined to fit through pre-cut, holes in a manifold plate. The clearance between the outer diameter of the tail and the inner diameter of the hole in the manifold plate is such that the tail will fit easily through the hole. The tail has positioning members on its outer diameter that when inserted fully into the hole in the manifold the positioning members compress to an interference fit which will hold the coupling in a ~~nominal~~ normal, i.e., substantially perpendicular, position to the manifold plate. Additionally, the retaining ring that holds the tail in place in the hole in the manifold plate can also have a positioning member that compresses against the manifold plate opposite of the male or female end of the coupling to hold the coupling in a nominal position to the plate.

[00015] In FIG. 2, a first embodiment of the undersea hydraulic coupling of the present invention is shown. FIG. 2 shows the male member of the coupling and its attachment to a manifold plate. The manifold plate 16 can be any thickness, but preferred standard thicknesses are 1 ¼ inch or 2 inch. The male member 14 of the hydraulic coupling is shown as having tail 18 which extends through manifold plate 16. Once the tail 18 is inserted through the manifold plate 16, it is held in place by retainer ring 20, which is held in locking position by snap ring 22 that fits into groove 24 that extends radially around the circumference of the tail 18. Of course, the retaining ring 20 could be threaded on or held in place through any number of alternatives that will be apparent to those of skill in the art. As can be seen in FIG. 2, there is a radial space 26 between the body of the tail 18 and the inner bore of the hole in manifold plate 16. Preferably the radial space 26 will be in the range of 0.025-0.050 inches. As will be further seen, the retainer ring 20 is not in tight fit with manifold plate 16, leaving gap 28 as well. Gap 28 is also preferably in the range of 0.025-0.050 inches. Because of radial space 26 and gap 28, the male member 14 may shift so as not to be ~~nominal~~ normal (substantially

perpendicular) to the manifold plate 16. This will be particularly true due to cantilevering when plate 16 is turned vertically so that male member 14 extends horizontally therefrom. The present embodiment prevents the tail 18 of the male member 14 from shifting, even when extending horizontally, through the use of substantially rigid positioning members 30 extending around the circumference of the tail 18 in grooves cut into the tail 18. The positioning members 30 are preferably elasomeric o-rings. The o-rings 30 have enough rigidity to hold the male member 14 ~~nominal~~ normal to the manifold plate 16. The end of the tail 36 of female member 32 that extends through manifold plate 34 is used to connect to hydraulic lines.

[00016] The o-rings of this embodiment are one type of a substantially rigid positioning member. The o-rings are not for the purpose of sealing, and are not required to main seal integrity. As those of skill in the art will note, the o-rings 30 of this embodiment may be replaced with any element that provides enough give to allow the tail 18 to be inserted, but enough rigidity to hold the male member 14 ~~nominal~~ nominally perpendicular to the manifold plate. Additionally, rather than having o-rings around the circumference of the tail 18, there could be fins that run longitudinally on the tail or any other suitable arrangement.

[00017] FIG. 3 shows the female member that mates with the male member of FIG. 2 and the tail structure of the female member that connects through its manifold plate. The female member 32 connects through manifold plate 34. The manifold plate 34 can be any thickness, but preferred standard thicknesses are 1 ¼ inch or 2 inch. The [[male]] female member 32 of the hydraulic coupling is shown as having tail 36 which extends through manifold plate 34. Once the tail 36 is inserted through the manifold plate 34, it is held in place by retainer ring 38, which is held in locking position by snap ring 40 that fits into groove 42 that extends radially around the circumference of the tail 36. In FIG. 3, as there was in FIG. 2, there is a radial space 44 between the body of the tail 36 and the inner bore of the hole in manifold plate 34. Preferably the radial space 44 will be in the range of 0.025-0.050 inches. As will be further seen, the retainer ring 38 is not in tight fit with manifold plate 34, leaving gap 46 as well. Preferably gap 46 is in the range of 0.025

– 0.050 inches. Because of radial space 44 and gap 46, the female member 32 may shift in the manifold plate 34 so as not to be ~~nominal~~ normal (substantially perpendicular) to the plate. The present embodiment prevents the tail 36 of the female member 32 from shifting, even when extending horizontally, through the use of substantially rigid positioning members 48 extending around the circumference of the tail 36 in grooves cut into the tail 36. The substantially rigid positioning members 48 are preferably elastomeric o-rings that have enough rigidity to hold the female member 32 ~~nominal~~ normal to the manifold plate 34. The end of the tail 36 of female member 32 that extends through manifold plate 34 is used to connect to hydraulic lines.

[00018] FIG. 4 shows another embodiment of the present invention showing a male member of a hydraulic coupling attached to a hydraulic plate as shown in FIG. 2 with the addition of an additional substantially rigid positioning member associated with the retainer ring. In FIG. 4, components that are the same as FIG. 2 bear the same component number as those of FIG. 2. In FIG. 4, in addition to o-rings 30 that help hold the male member 14 in a ~~nominal~~ nominally perpendicular relationship with hydraulic plate 16, a modified retainer ring 50 is used. The modified retainer ring 50 has a groove cut into the radial face that engages with the manifold plate 16. Another substantially rigid positioning member 52, preferably an o-ring, is inserted in the groove in the modified retainer ring 50. The o-ring 52 is in contact with the modified retainer ring 50 and the manifold plate 16 so as to further resist cantilevering of the male member 14.